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HPC Training Series

An introduction to adjoint methods and the shape and topology optimization workflow of OpenFOAM for CFD-based optimization

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What is Gradient-based Optimization?

Optimization Methods in CFD:

Improve the performance of an aerodynamic shape

Quantity describing the performance:

The objective function J (e.g. drag force exerted on a car) → computed through CFD

How are going to affect the objective function?:

By changing the values of the so-called design variables, \vec{b} .
For instance, control points affecting the shape of the car

How are we going to update the values of the design variables?:

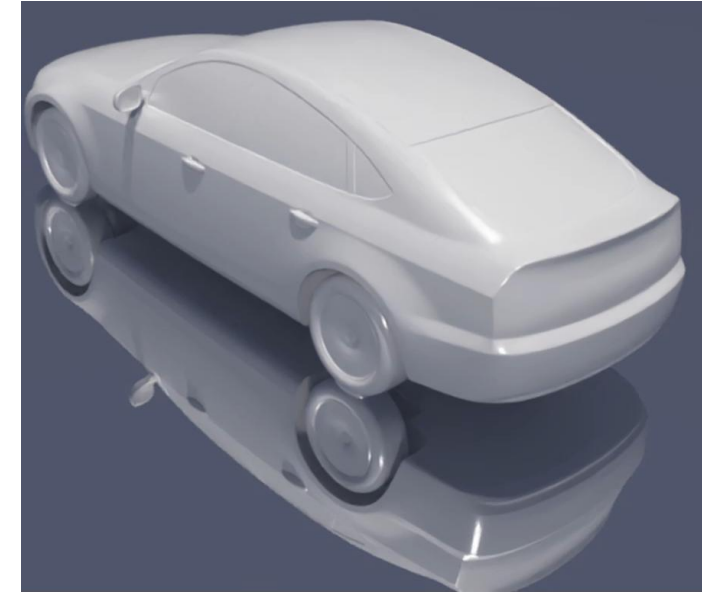
Gradient-free Methods:

Require only the computation of J .
*See the flow solver (or any evaluation tool)
as a black box*

Gradient-based Methods:

Require the computation of J and $(dJ/d\vec{b})$
Potentially need access to the source code

How is this computed?





How can $dJ/d\vec{b}$ be computed?

The simplest way → Finite Differences

- ✓ Does not require the development of any additional s/w.
Relies only on the flow solver
- ✗ Its cost scales with the number of design variables N
- ✗ Sensitive to the choice of the ϵ step

$$\frac{dJ}{db_i} = \frac{J(b_i + \epsilon) - J(b_i)}{\epsilon}$$

The hard way → The adjoint method

- ✗ Requires a new mathematical development and programming if the flow problem or the objective function changes
- ✓ Has a cost that is independent of the number of the design variables. Ideal for expensive industrial problems with many design variables
- ✓ All components of $dJ/d\vec{b}$ are computed at the cost of only an additional set of PDEs → the adjoint equations

Review paper:

<https://doi.org/10.1007/s11831-014-9141-9>



The flow and adjoint PDEs



Continuity

$$R^p = \frac{\partial v_j}{\partial x_j} = 0$$

Momentum

$$R_i^v = v_j \frac{\partial v_i}{\partial x_j} + \frac{\partial p}{\partial x_i} - \frac{\partial \tau_{ij}}{\partial x_j} = 0, \quad i = 1, 2, 3$$

Spalart-Allmaras

$$R^{\tilde{\nu}} = v_j \frac{\partial \tilde{\nu}}{\partial x_j} - \frac{\partial}{\partial x_j} \left(\frac{(\nu + \tilde{\nu})}{\sigma} \frac{\partial \tilde{\nu}}{\partial x_j} \right) - \frac{c_{b2}}{\sigma} \left(\frac{\partial \tilde{\nu}}{\partial x_j} \right)^2 + \tilde{\nu} (D(\tilde{\nu}, y) - P(\tilde{\nu}, y)) = 0$$

The adjoint PDEs

Adjoint Continuity

$$R^q = \frac{\partial u_i}{\partial x_i} = 0$$

Adjoint Momentum

$$R_i^u = -\frac{\partial (v_j u_i)}{\partial x_j} + u_j \frac{\partial v_j}{\partial x_i} + \frac{\partial q}{\partial x_i} - \frac{\partial \tau_{ij}^\alpha}{\partial x_j} + \frac{\partial}{\partial x_j} \left(\frac{C_{\tilde{S}}}{S} \tilde{\nu}_a \tilde{\nu} \left(\frac{\partial v_i}{\partial x_j} - \frac{\partial v_j}{\partial x_i} \right) \right) = 0, \quad i = 1, 2, 3$$

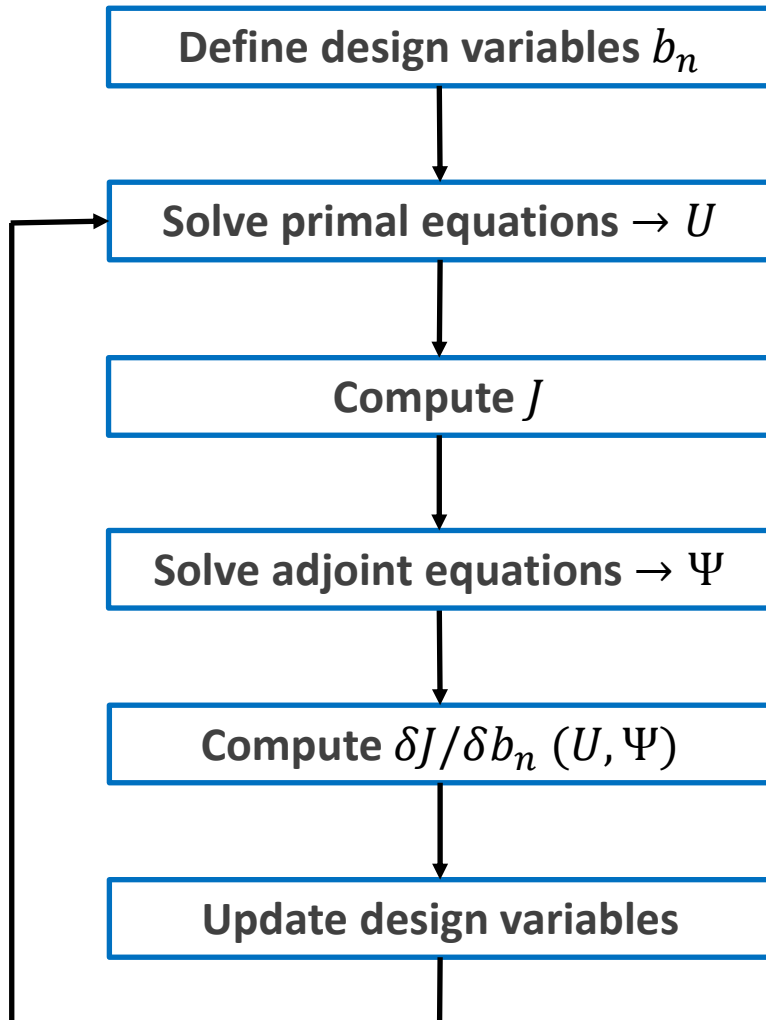
Adjoint Spalart-Allmaras

$$R^{\tilde{\nu}_a} = -\frac{\partial (v_j \tilde{\nu}_a)}{\partial x_j} - \frac{\partial}{\partial x_j} \left(\frac{\nu + \tilde{\nu}}{\sigma} \frac{\partial \tilde{\nu}}{\partial x_j} \right) + \frac{1}{\sigma} \frac{\partial \tilde{\nu}}{\partial x_j} \frac{\partial \tilde{\nu}_a}{\partial x_j} + 2 \frac{c_{b2}}{\sigma} \frac{\partial}{\partial x_j} \left(\tilde{\nu}_a \frac{\partial \tilde{\nu}}{\partial x_j} \right) + C_{\tilde{\nu}} \tilde{\nu} \tilde{\nu}_a$$

$$+ \tilde{\nu}_a (D(\tilde{\nu}, y) - P(\tilde{\nu}, y)) + \frac{\partial u_i}{\partial x_j} \left(\frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \frac{\delta \nu_t}{\delta \tilde{\nu}} = 0$$



An Adjoint-based Optimization Loop



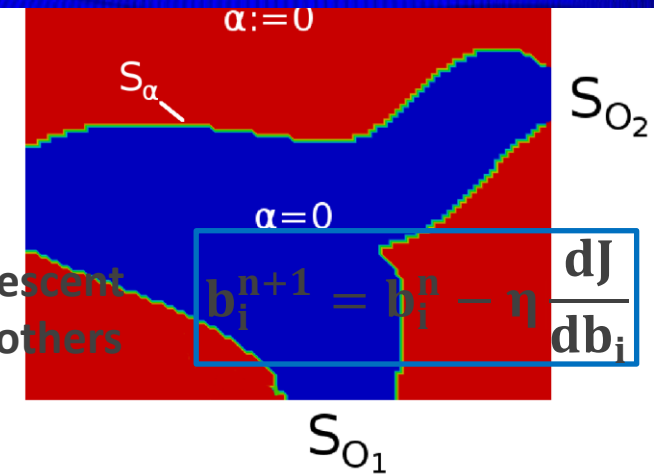
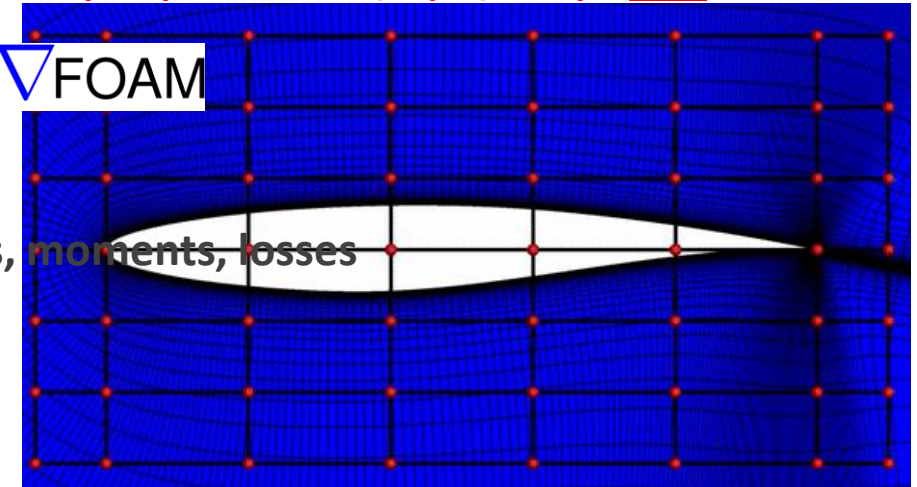
⇒ Effect on the flow equations?

⇒ OpenFOAM

⇒ Forces, moments, losses

⇒ Steepest descent and many others

Shape optimization (ShpO): shape and mesh



$$b_i^{n+1} = b_i^n - \eta \frac{dJ}{db_i}$$



adjointOptimisationFoam: an adjoint-based optimization framework in OpenFOAM



- An all-in-one OpenFOAM executable implementing an integrated, gradient-based optimization workflow
- Product of a 17 years of development at PCOpt/NTUA
- Integrated into the official OpenFOAM version in collaboration with OpenCFD in 2019
- User manual:
https://www.openfoam.com/documentation/files/adjointOptimisationFoamManual_v2312.pdf
Covers all functionality up until v2506

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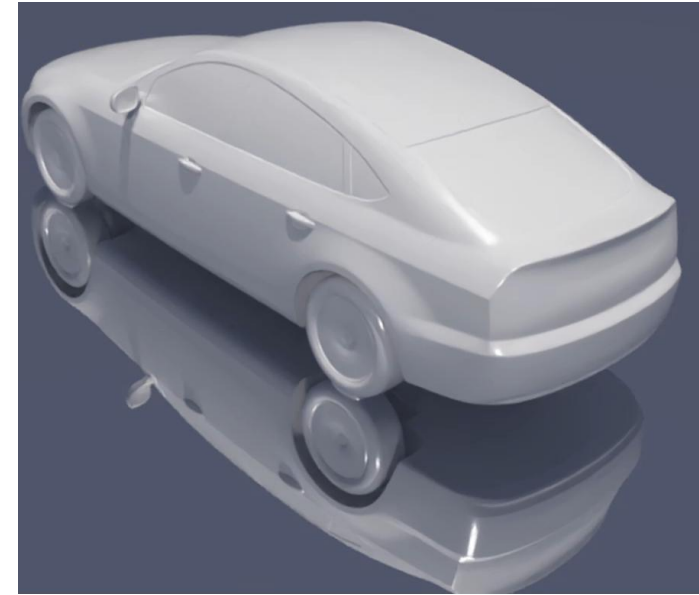
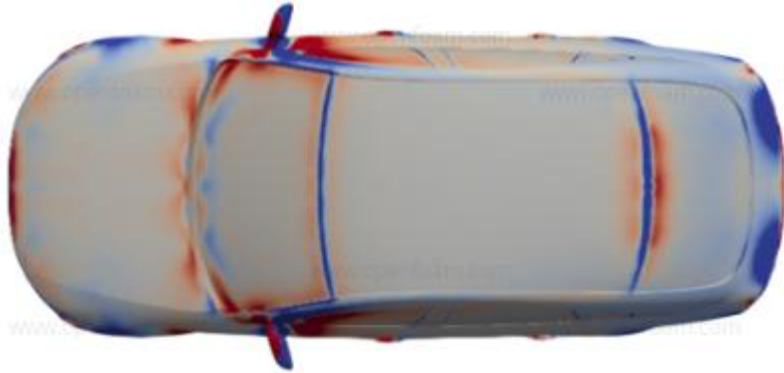


Current status of adjointOptimisationFoam

OpenFOAM version	Features
v1906	<ul style="list-style-type: none"> • Adjoint to incompressible, steady-state flows • Differentiation of the Spalart-Allmaras turbulence model • Computation of sensitivity maps
v1912	<ul style="list-style-type: none"> • Surface and volume parameterization using volumetric B-Splines • Automated shape optimization loops
v2006	<ul style="list-style-type: none"> • New objective function related to the qualitative evaluation and minimization of noise • Sensitivity contributions from rotating boundaries
v2112	<ul style="list-style-type: none"> • Smoothing of sensitivity maps
v2206	<ul style="list-style-type: none"> • Adjoint to the k-ω SST turbulence model
v2212	<ul style="list-style-type: none"> • Objective functions for internal aerodynamics (flow rate, flow rate distribution, uniformity, power losses)
v2312	<ul style="list-style-type: none"> • Topology optimization
2406-2506	<ul style="list-style-type: none"> • Incremental improvements



Sensitivity maps and Shape Optimization (ShpO)



Sensitivity maps:

- The derivative of J w.r.t. the normal displacement of the boundary nodes
- No optimization loop; only 1 flow + 1 adjoint solution
- Identify the areas with a high optimization potential → Intense colors
- Identify the favorable displacement direction →

Blue: move surface inwards

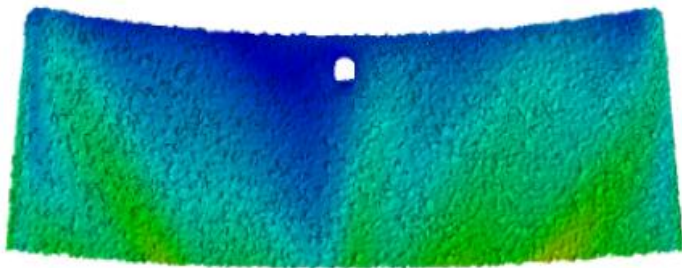
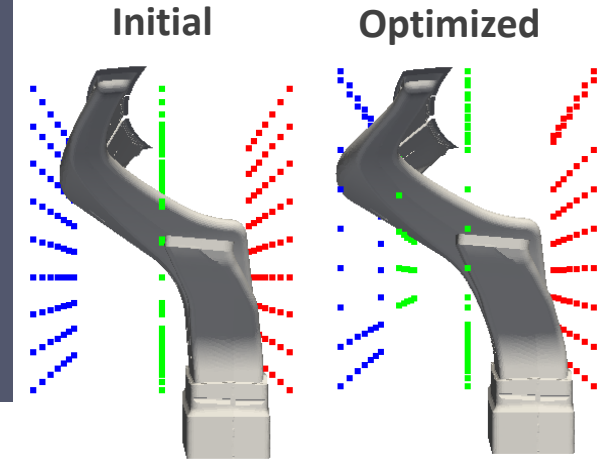
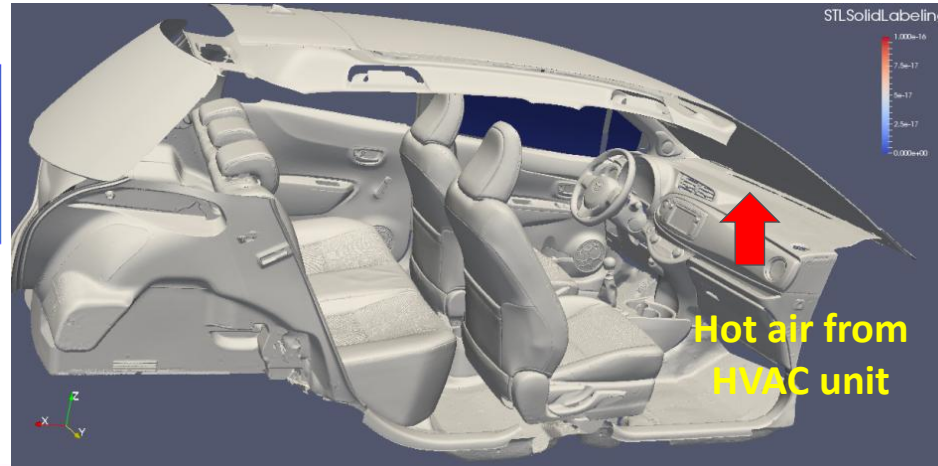
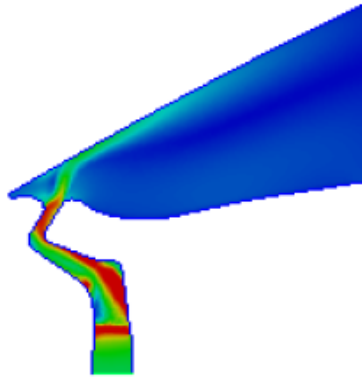
Red: move surface outwards

Shape optimization:

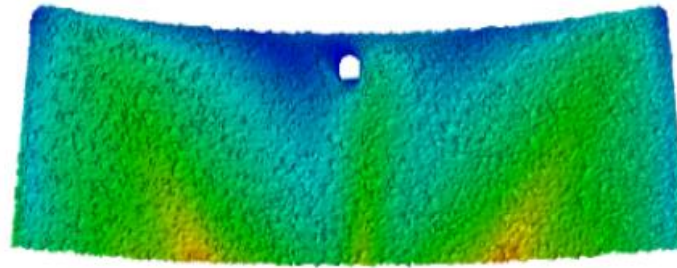
- An actual optimization loop is performed
- In each optimization cycle, the shape is updated, followed by the update of the internal grid nodes
- Each cycle has a cost of 1 flow + 1 adjoint solution
- Usually, a small number of cycles is required to reach convergence (< 20)



ShpO of the Defroster Nozzle of the HVAC unit of a Car



Initial



Optimized

$$J_V = \frac{1}{2} \int_{\Omega'} (v - v^{target})^2 d\Omega$$

ShpO of the defroster nozzle of the HVAC unit of a TOYOTA passenger car, to shorten the time for dispelling condensation or frost on the windshield in the most uniform way. To this end, a certain air velocity close to the windshield must be reached. The optimized geometry was manufactured (3D printing) and submitted to a defrost test in the TME's climate chamber (@ -20°), leading to 15% less windshield defrost time.

Green areas in the velocity isolines' plot on the windshield correspond to v^{target} .





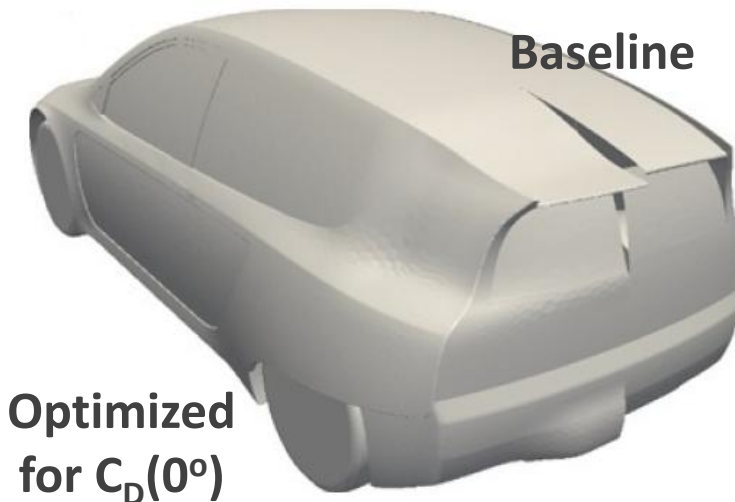
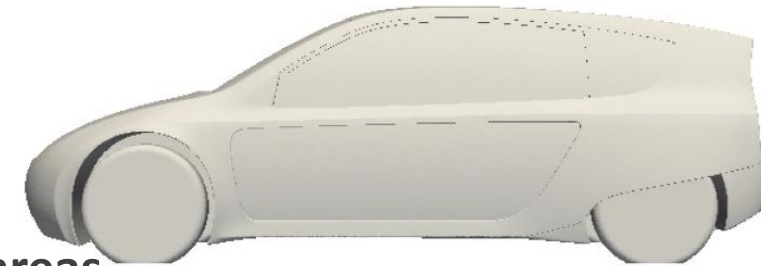
Multi-Point Aerodynamic ShpO of a Concept Car

- ShpO of an ultra-lightweight vehicle, designed by the Toyota Aerodynamic Dept., for making it less sensitive to side-wind (30° from the port side; min. yaw moment), while maintaining a very low drag at 0° (longitudinal wind).

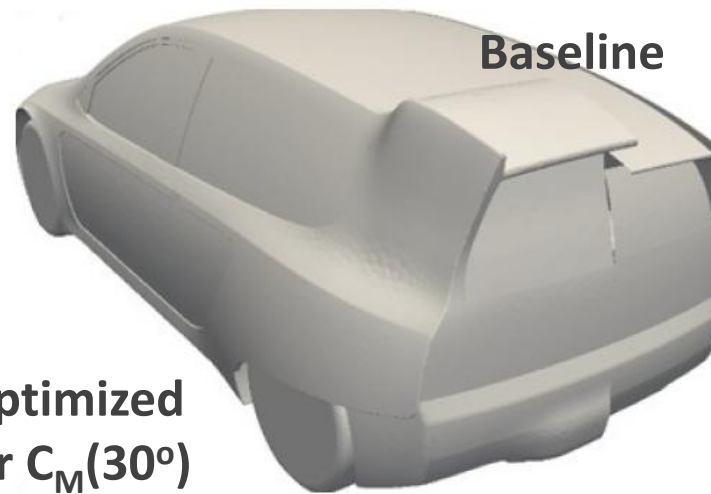
- Objective function:

$$J = \omega_D C_D^{0^\circ} + \omega_M C_M^{30^\circ}$$

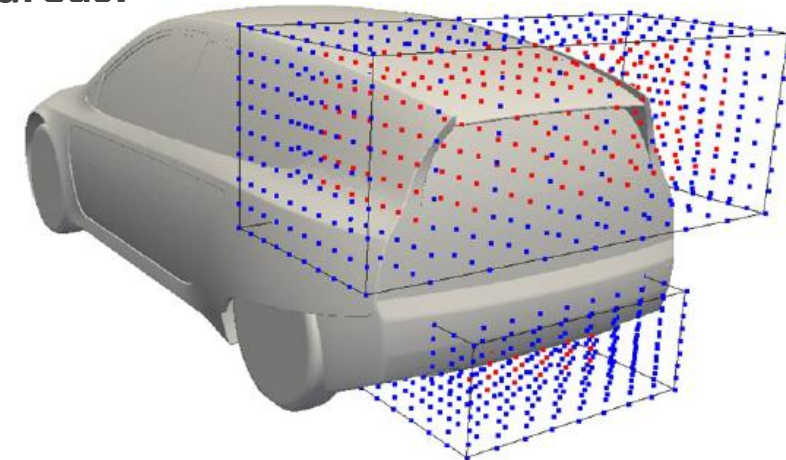
- Pareto front computed by optimizing with different values of (ω_D, ω_M) .
- Two simultaneously acting morphing boxes at the spoiler and diffuser areas.



Optimized for $C_D(0^\circ)$

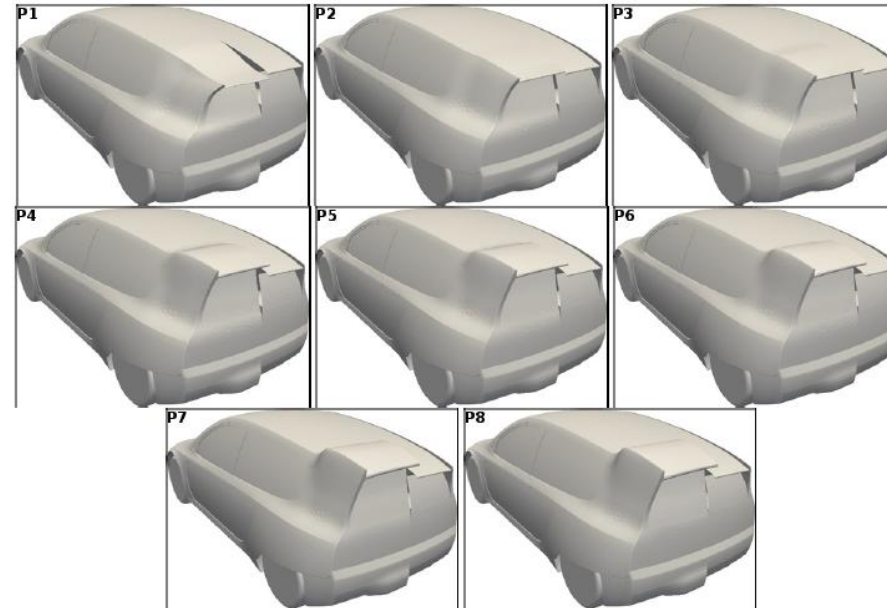
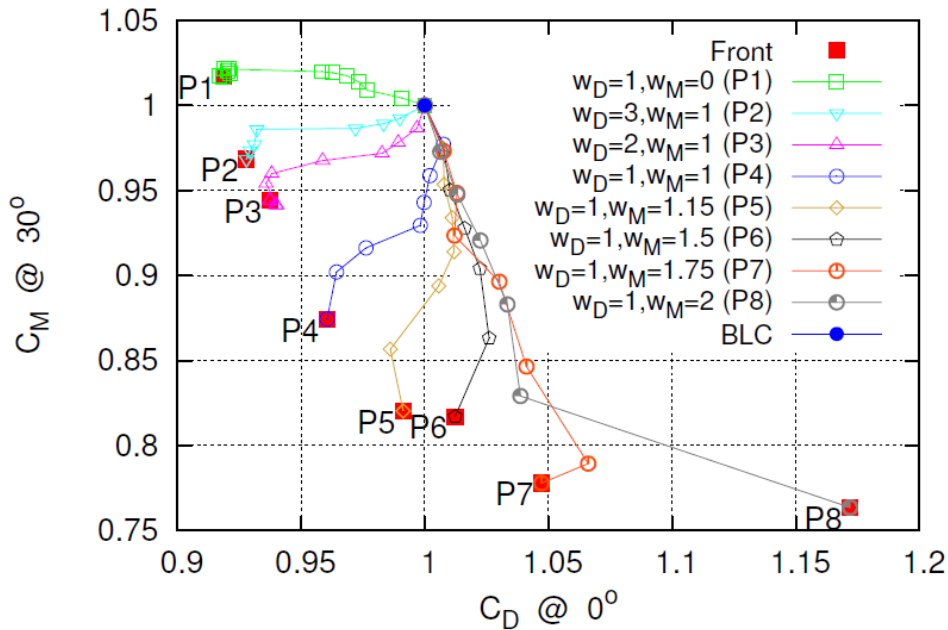


Optimized for $C_M(30^\circ)$





Multi-Point Aerodynamic ShpO of a Concept Car



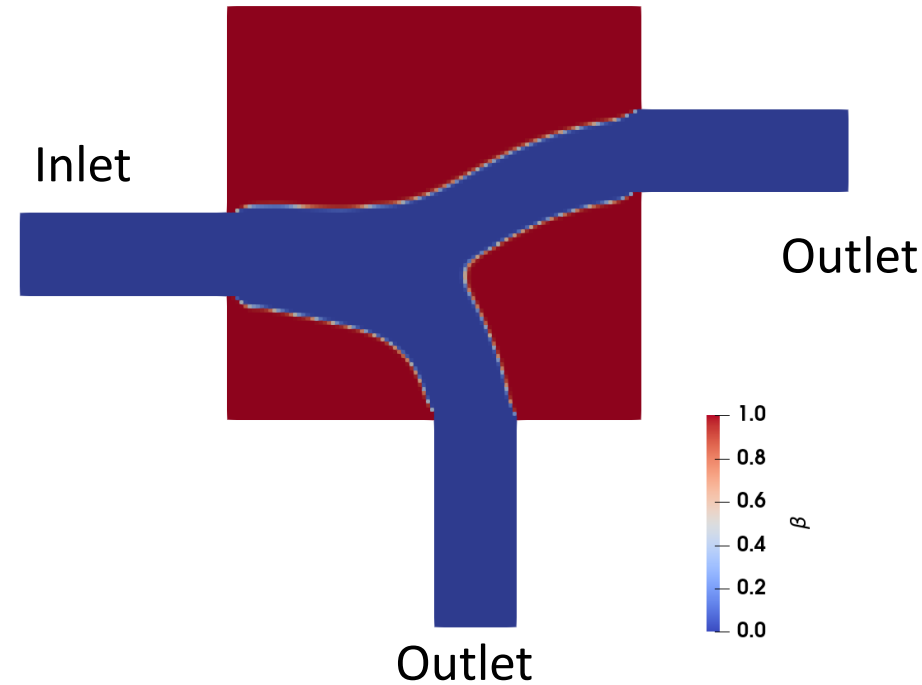
RANS-based ShpO using the adjoint to the Spalart-Allmaras model (with wall functions). Optimized geometries (port side) compared to the baseline (starboard side). C_D reduction at 0° results from a lowered spoiler, boat-tailing and a prolonged and widened diffuser. C_M reduction at 30° comes mainly from the increased spoiler height and the slight widening of the car; these increase pressure on the port side and decrease it on the starboard side to counter-balance the yaw moment due to side-wind.

Structural and Multidisciplinary Optimization, 59(2): 675–694, 2019.



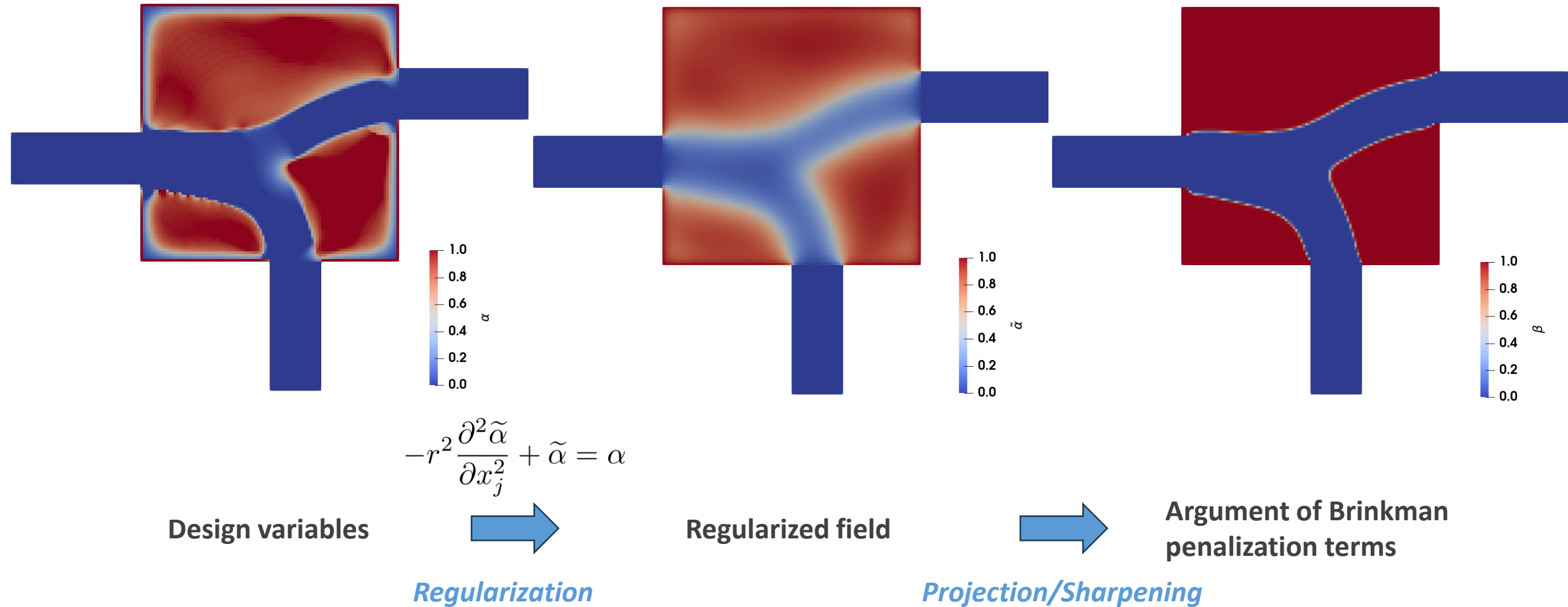
Topology Optimization (TopO)

- Primarily used for early design of duct systems with known inlets/outlets
- No shape parameterization
- Counter-productive cells are solidified through a source term in the flow equations
- $\beta \sim 1$, solidified domain; theoretically, impermeable to flow
- $\beta = 0$, flow domain
- Topology optimization: seeks optimal β fluid/solid identifier to minimize an objective function and satisfy the given constraints
- Number of design variables = Number of mesh cells
- Sensitivity derivatives computed with (continuous) adjoint





From design variables to Brinkman penalization terms





Primal Equations

- Most general case examined: Navier-Stokes equations & the Spalart-Allmaras model for turbulent flows:

Brinkman penalization terms

$$R^p = \frac{\partial v_i}{\partial x_i} = 0$$

$$R_i^v = v_j \frac{\partial v_i}{\partial x_j} - \frac{\partial \tau_{ij}}{\partial x_j} + \frac{\partial p}{\partial x_i} + \underline{\beta_{max} I_v(\beta) v_i} = 0$$

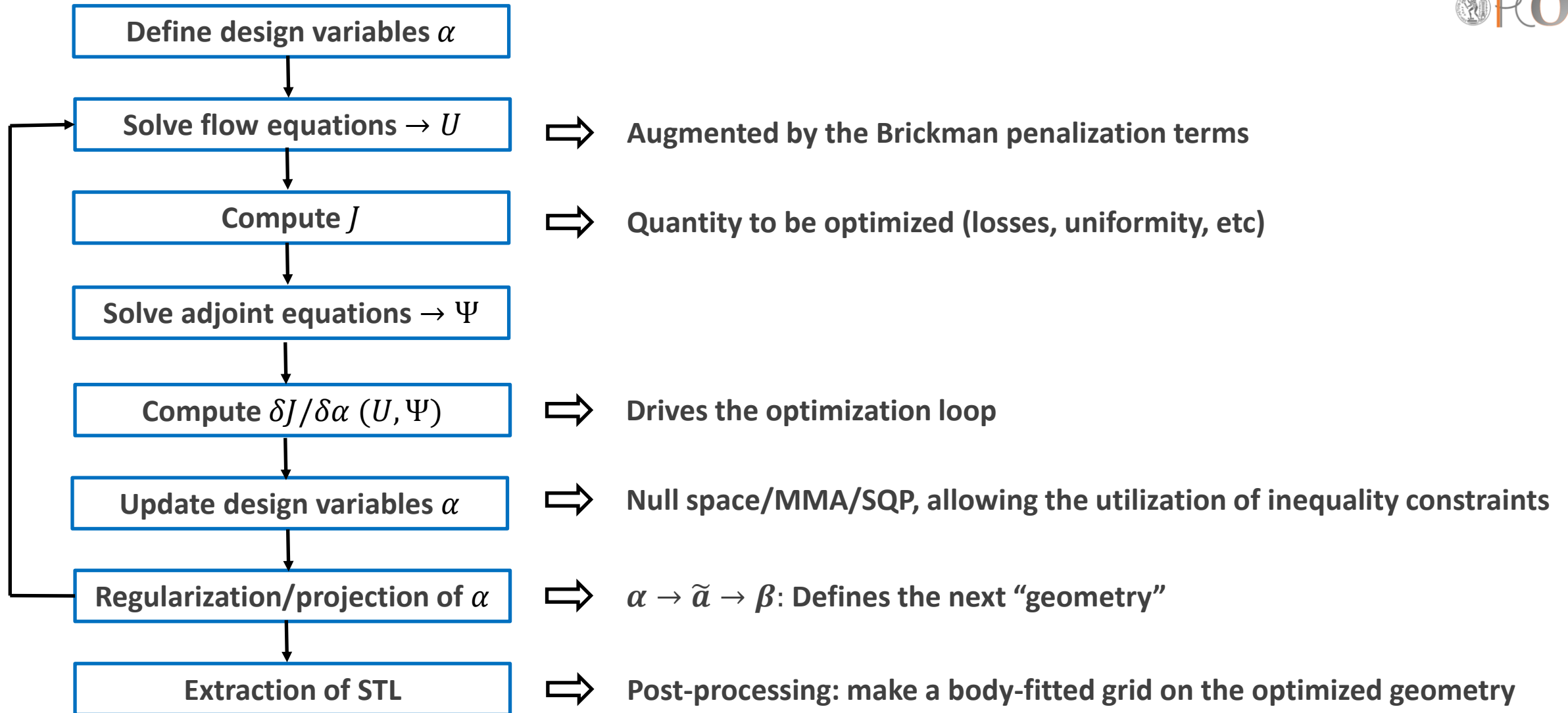
$$R^{\tilde{v}} = v_j \frac{\partial \tilde{v}}{\partial x_j} - \frac{\partial}{\partial x_j} \left[\left(\nu + \frac{\tilde{v}}{\sigma} \right) \frac{\partial \tilde{v}}{\partial x_j} \right] - \frac{c_{b2}}{\sigma} \left(\frac{\partial \tilde{v}}{\partial x_j} \right)^2 - \tilde{v} \mathcal{P}(\tilde{v}) + \tilde{v} \mathcal{D}(\tilde{v}) + \underline{\beta_{max} I_{\tilde{v}}(\beta) \tilde{v}} = 0$$

$$R^\Delta = \frac{\partial}{\partial x_j} \left(\Delta \frac{\partial \Delta}{\partial x_j} \right) - \Delta \frac{\partial^2 \Delta}{\partial x_j^2} - 1 + \underline{\beta_{max} I_\Delta(\beta) \Delta} = 0$$

- β is related to the design variable field α
- β_{max} is a dimensioned constant ensuring that the variable computed by the PDE tends to zero when β is close to unity.



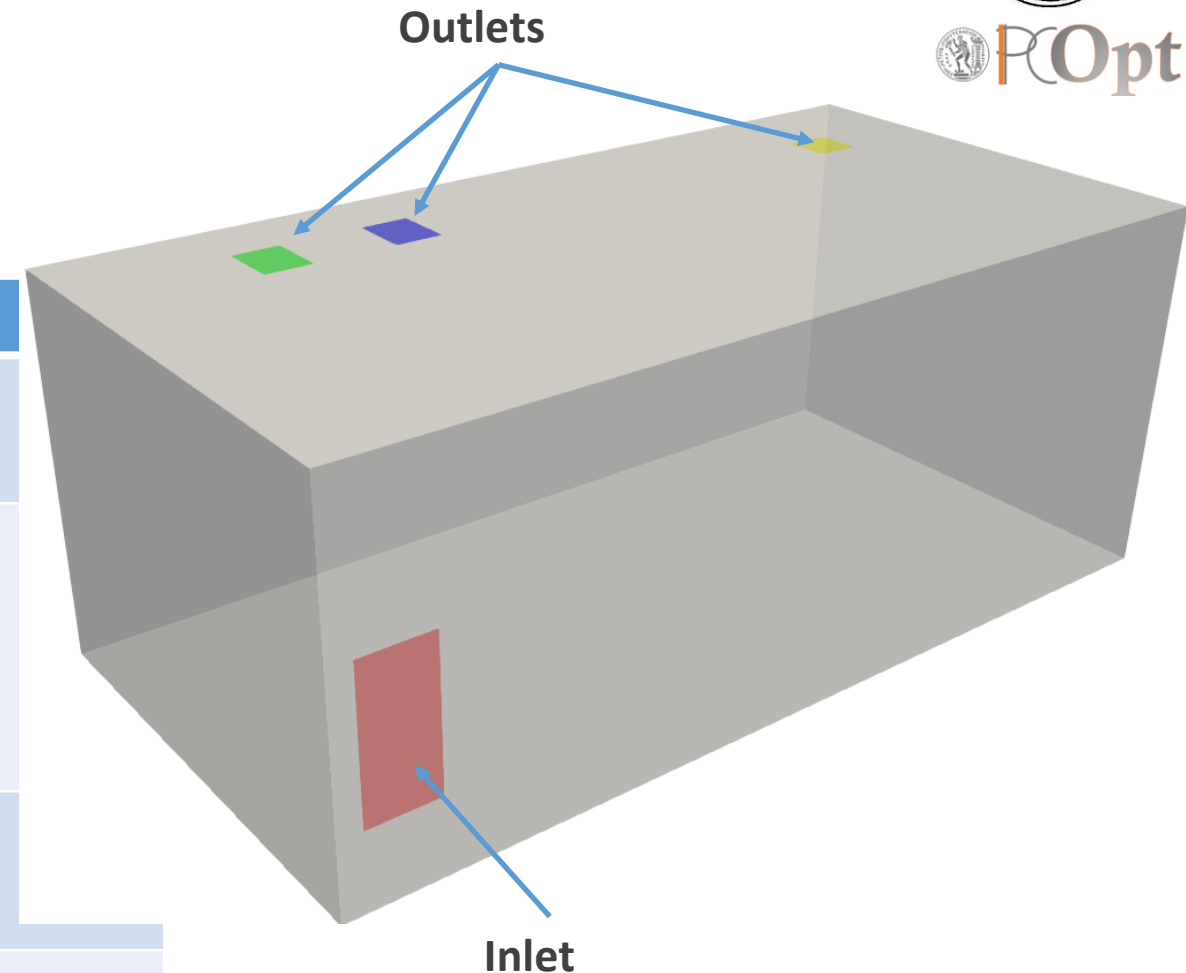
Topology optimization loop





3D TopO, Foot channel HVAC duct

- $Re = 1.3 \times 10^5$ (turbulent flow, SA model)
- 1.1×10^5 design variables
- Multiple available objective functions

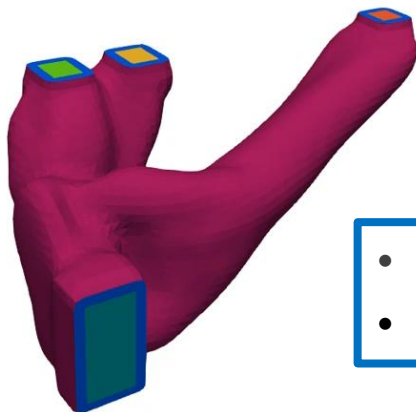
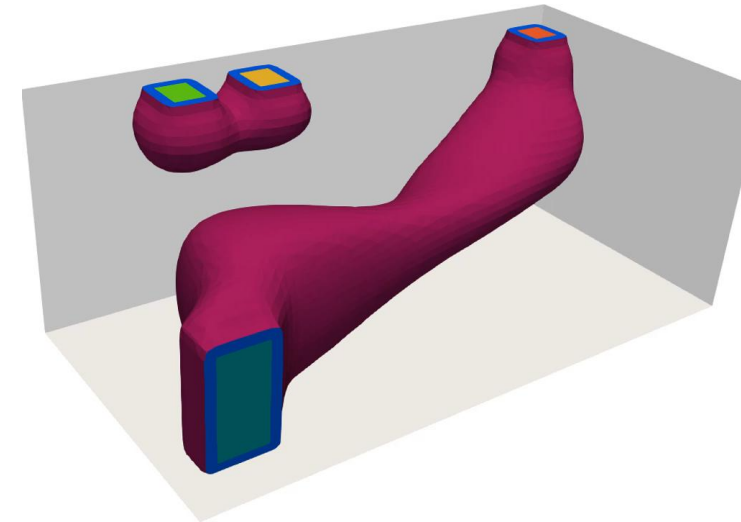
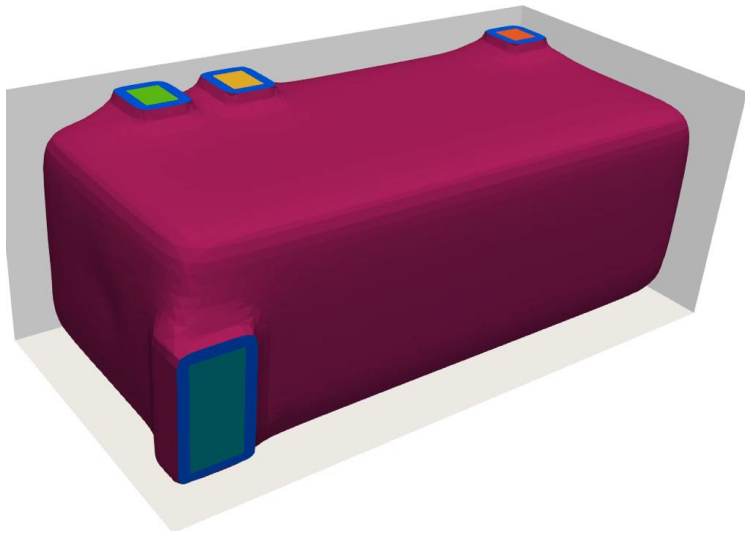


Objective	Formula
Total pressure losses	$J_{pt} = - \int_{S_{I,O}} p_t v_i n_i dS$
Flow rate partition	$J_m = 0.5 \sum_l (m_l - m_l^{tar})^2$ $m_l = - \int_{S_{O,l}} v_i n_i dS / \int_{S_I} v_i n_i dS$
Non-uniformity index	$J_u = 0.5 \sum_l \int_{S_l} (v_i - v_i^{mean})^2 dS$
Fluid volume	$V_F = \frac{\int_{\Omega} (1 - \beta) d\Omega}{\int_{\Omega} d\Omega}$

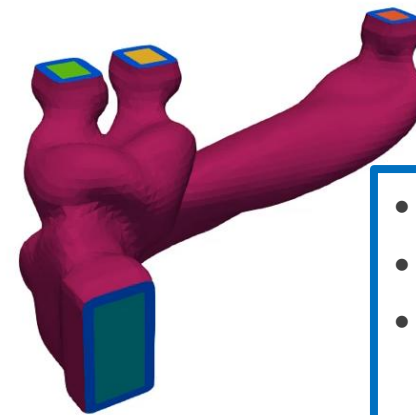


3D TopO, Foot channel HVAC duct

G1



- Min. J_{p_t}
- $V_F < V_F^{tar}$

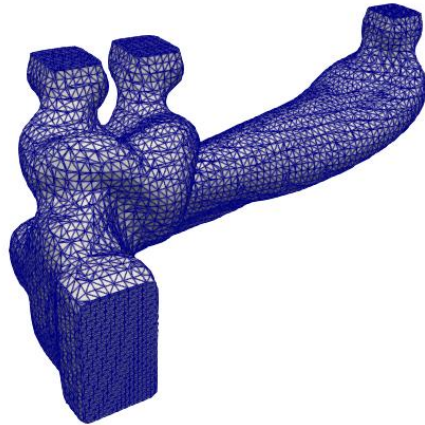


- Min. non – uniformity
- s.t.
- Equally-distribute flow rate
- $J_{p_t} < J_{p_t}^{tar}$
- $V_F < V_F^{tar}$

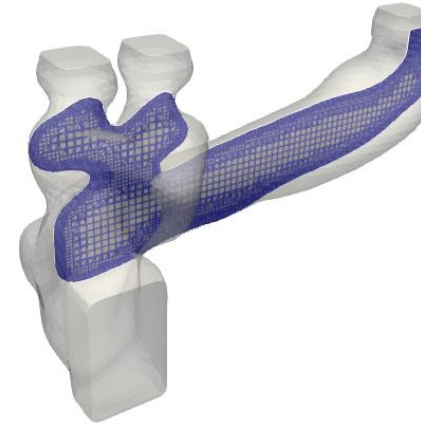


Foot channel HVAC duct: Re-evaluation on body-fitted meshes

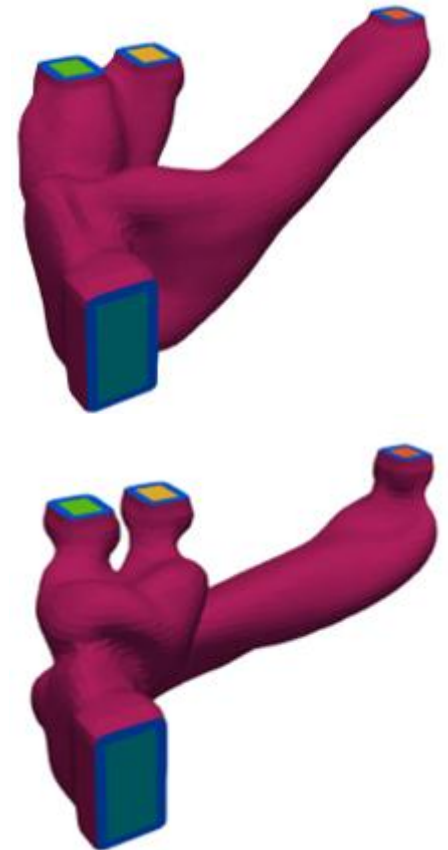
STL of the optimized geometry extracted by the topO code



snappyHexMesh



Body-fitted evaluation



Geometry	J_{p_t}		Mass distribution		J_u	
	TopO	Body-fitted	TopO	Body-fitted	TopO	Body-fitted
G1	15.43	13.69	33/34.5/32.5	34/36/30	541	898
G2	22	16.7	33/34/33	32/34/34	354	873



Publicly Available Tools in OpenFOAM:

The latest version of the software can be downloaded from

<https://develop.openfoam.com/Development/openfoam>

The development branch can be found in

<https://develop.openfoam.com/Development/openfoam/-/tree/develop>

Extensive user-guide is available at

https://www.openfoam.com/documentation/files/adjointOptimisationFoamManual_v2312.pdf