



EURO  
Greece

# HPC Training Series

*Introduction to OpenFOAM:  
OpenFOAM's history, main features and case setup*

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# Main objectives of the course

- **Familiarization with CFD and OpenFOAM**
  - Short history
  - Capabilities and characteristics
  - Case setup
- **An introduction to managing jobs in an HPC environment**
  - Simulated HPC environment using the SLURM job submission system
  - Demonstration of a CFD job submission in the ARIS HPC system of GRNET

# What is OpenFOAM?

- **Mainly a toolbox for solving Partial Differential Equations (PDEs)**
- **Started out in the Imperial College of London, during the 90s**
- **Started being distributed as open-source software during 2004**
- **Widely used by a variety of industries (automotive, turbomachinery, chemicals, paper, energy, etc)**
- **A wide variety of applications, mainly focused on Computational Fluid Dynamics (CFD), but not exclusively (some support for structural mechanics, finance, etc)**

# What is OpenFOAM?

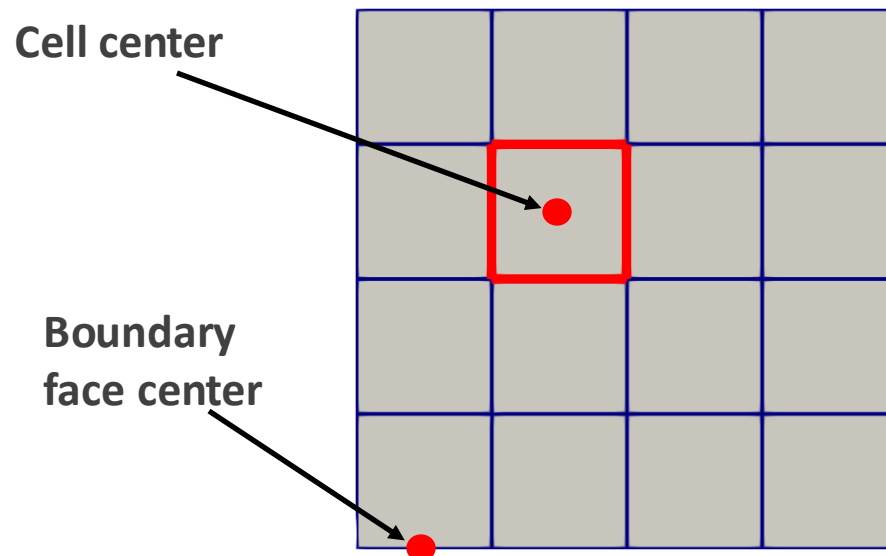
- **Mainly a toolbox for solving Partial Differential Equations (PDEs)**
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# What is required by the user

- As an open-source s/w, its “natural” environment lays in Linux
- Windows support through the Windows Subsystem for Linux (WSL)
  - We will be using OpenFOAM v2312  
<https://www.openfoam.com/news/main-news/openfoam-v2312>
- No Graphical User Interface (GUI!)
  - Setup and navigation of cases using the Linux Command Line Interface (CLI)
    - Some basics skills are needed (navigation through the file system, creating/copying/deleting files and folders, editing files through a text editor)
    - Some frequent CLI commands can be found [here](#)
    - Some useful CLI features of OpenFOAM can be found [here](#)
- Steep learning curve
  - A number of files need to be setup before running a case
  - A lot of freedom for the user
    - ✓ A truly versatile s/w with very few constraints
    - ✗ Freedom to setup a case with little physical meaning
  - It requires from the user to understand the problem they are trying to solve

## Basic OpenFOAM characteristics

- Based on the finite volumes method (flow equations integrated over control/finite volumes)
- Written in C++, taking full advantage of the object oriented aspects of the language (~3M lines of code)
- Based on a cell centered discretization of the flow equations



Flow quantities are computed/stored at the cell centers and boundary conditions are imposed on the boundary face centers

# Flow equations for incompressible fluids

A plethora of solvers exists (incompressible, compressible, single- and multi-phase, mixtures, etc).  
 Today, we are going to focus on incompressible fluids (constant fluid density)

## Navier-Stokes equations

$$R^p = \frac{\partial v_j}{\partial x_j} = 0 \quad \text{Flow velocity}$$

Continuity equation: Conservation of mass

$$R_i^v = \underbrace{\frac{\partial(v_j v_i)}{\partial x_j}}_{m\alpha} - \frac{\partial}{\partial x_j} \left[ \underbrace{(\nu + \nu_t)}_{\text{Bulk viscosity}} \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

$\sum F$   
 Pressure

Momentum equations: Newton's 2<sup>nd</sup> law of motion

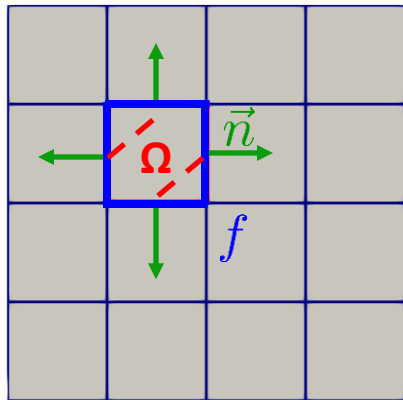
# The Finite Volume Method in brief

$$R^p = \frac{\partial v_j}{\partial x_j} = 0$$

Continuity equation: Conservation of mass

$$R_i^v = \frac{\partial(v_j v_i)}{\partial x_j} - \frac{\partial}{\partial x_j} \left[ (\nu + \nu_t) \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

Momentum equations: Newton's 2<sup>nd</sup> law of motion



FVM = **integration** of the flow equations over a control (finite) volume ( $\Omega$ )

Gauss divergence theorem

Discretization

Volume flow rate (m<sup>3</sup>/s)

$$\int_{\Omega} \frac{\partial v_j}{\partial x_j} d\Omega = \int_S v_j n_j dS = \sum_f \underbrace{v_j^f n_j^f}_{\text{Volume flow rate (m}^3\text{/s)}} dS^f$$

Interpolating to the boundaries of the finite volume is a crucial part of FVM



# Segregated vs coupled solution of the flow equations

OpenFOAM solves its equations in a segregated manner

## Coupled solution

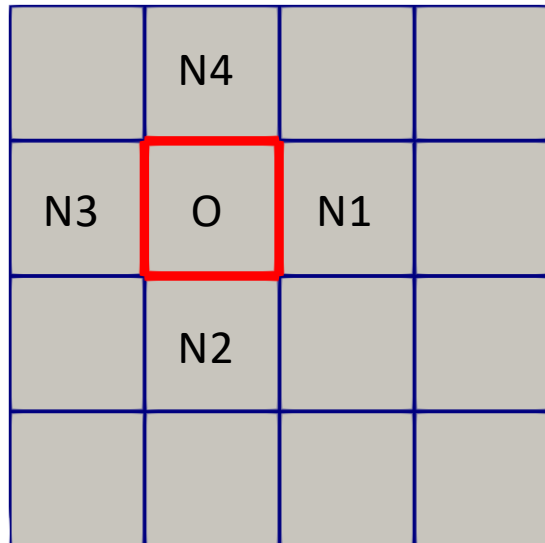
$$\begin{bmatrix} \mathbf{A}_{U_{xx}}(\tilde{\mathbf{U}}) & \mathbf{A}_{U_{xy}}(\tilde{\mathbf{U}}) & \mathbf{A}_{U_{xp}}(\tilde{\mathbf{U}}) \\ \mathbf{A}_{U_{yx}}(\tilde{\mathbf{U}}) & \mathbf{A}_{U_{yy}}(\tilde{\mathbf{U}}) & \mathbf{A}_{U_{yp}}(\tilde{\mathbf{U}}) \\ \mathbf{A}_{px}(\tilde{\mathbf{U}}) & \mathbf{A}_{py}(\tilde{\mathbf{U}}) & \mathbf{A}_{pp}(\tilde{\mathbf{U}}) \end{bmatrix} \begin{bmatrix} U_x \\ U_y \\ p \end{bmatrix} = \begin{bmatrix} b_x \\ b_y \\ b_p \end{bmatrix} \quad \vec{U} = \begin{bmatrix} U_x \\ U_y \\ p \end{bmatrix}$$

## Segregated solution

$$\begin{bmatrix} \mathbf{A}_{U_{xx}}(\tilde{\mathbf{U}}) & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{A}_{U_{yy}}(\tilde{\mathbf{U}}) & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{A}_{pp}(\tilde{\mathbf{U}}) \end{bmatrix} \begin{bmatrix} U_x \\ U_y \\ p \end{bmatrix} = \begin{bmatrix} b_x - \mathbf{A}_{U_{xy}}(\tilde{\mathbf{U}})U_y - \mathbf{A}_{U_{xp}}(\tilde{\mathbf{U}})p \\ b_y - \mathbf{A}_{U_{yx}}(\tilde{\mathbf{U}})U_x - \mathbf{A}_{U_{yp}}(\tilde{\mathbf{U}})p \\ b_p - \mathbf{A}_{px}(\tilde{\mathbf{U}})U_x - \mathbf{A}_{py}(\tilde{\mathbf{U}})U_y \end{bmatrix}$$

# Implicit discretization schemes

- Implicit discretization schemes for spatial and temporal derivatives



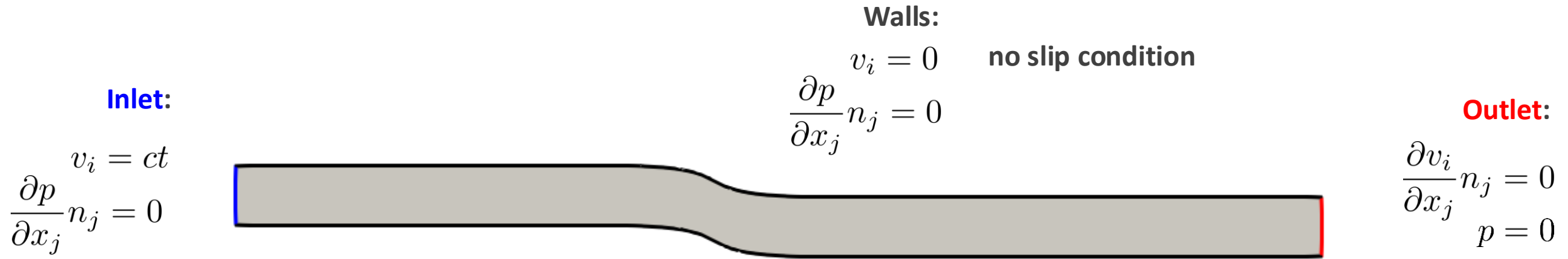
Discretization of the x component of the momentum equations on cell O, with neighbors N

$$R_{U_{x,O}} = a_O U_{x,O} + \sum a_N U_{x,N} - B_x(U_x, U_y, p) = 0$$

$$[A_{U_{xx},O}(\tilde{U}) \quad A_{U_{xx},N}(\tilde{U})] \begin{bmatrix} U_{x,O} \\ U_{x,N} \end{bmatrix} = [B_{x,O}]$$

$$\begin{bmatrix} A_{U_{xx}}(\tilde{U}) & 0 & 0 \\ 0 & A_{U_{yy}}(\tilde{U}) & 0 \\ 0 & 0 & A_{pp}(\tilde{U}) \end{bmatrix} \begin{bmatrix} U_x \\ U_y \\ p \end{bmatrix} = \begin{bmatrix} b_x - A_{U_{xy}}(\tilde{U})U_y - A_{U_{xp}}(\tilde{U})p \\ b_y - A_{U_{yx}}(\tilde{U})U_x - A_{U_{yp}}(\tilde{U})p \\ b_p - A_{px}(\tilde{U})U_x - A_{py}(\tilde{U})U_y \end{bmatrix}$$

# Boundary conditions



- Why impose a zero outlet pressure?

$$R_i^v = \frac{\partial(v_j v_i)}{\partial x_j} - \frac{\partial}{\partial x_j} \left[ (\nu + \nu_t) \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

The pressure appears only as a gradient in the flow equations ...

- The pressure field is computed wrt a reference pressure, usually imposed at the outlet
- The pressure field computed in incompressible flow simulations is actually the pressure divided by the constant fluid density

# Typical structure of an OpenFOAM case (1)

- Three main folders `0`, `constant`, `system`

`0`

- Setup of the boundary conditions and the flow initialization. Should include one file per flow field to be computed

`constant`

- `polyMesh` (folder)
  - The mesh folder. Meshes can be built using OpenFOAM's mesh generators (`blockMesh`, `snappyHexMesh`) or imported through preprocessing applications supporting the most popular mesh types
  - The `boundary` file includes the names and types of the patches of the mesh. Useful for setting up boundary conditions in the `0` folder
- `transportProperties`
  - Definition of basic fluid characteristics (e.g. viscosity)
- `turbulenceProperties`
  - Definition of the turbulence model

## Typical structure of an OpenFOAM case (2)

**system:** includes dictionaries controlling our simulation

- **controlDict**
  - A number of entries controlling the number of iterations to be executed, time-step (for unsteady runs), write interval, etc
  - Can also define functions, to be executed after each iteration or at a post-processing level
- **fvSolution**
  - Definition of the linear solvers, convergence criteria, relaxation factors
- **fvSchemes**
  - Definition of the schemes used to discretize terms in the flow equations
- **decomposeParDict** (optional, pre-processing)
  - Used to decompose the mesh into sub-domains, as a prerequisite for running in parallel
- **fvOptions** (optional)
  - Defines additional source terms for the flow/adjoint equations.

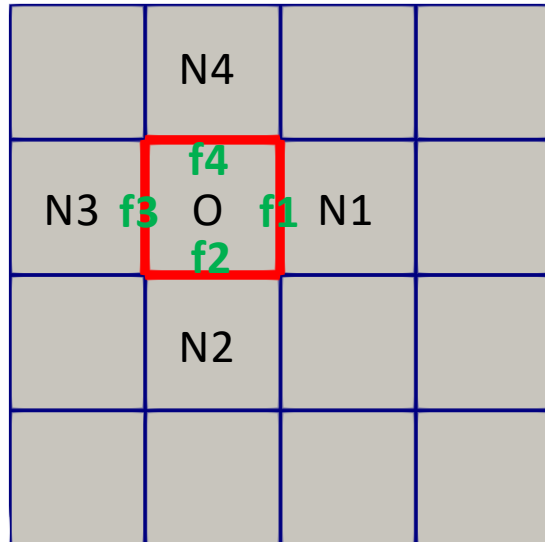
More details about the structure of an OpenFOAM case can be found [here](#)

## Our first OpenFOAM case

```
>> cd 2024_06_OF_training_EuroCC_Greece/01-secondOrder
```

- Go through the content of `constant`, `0` and `system`
- Use the `slurm.sh` and `Allclean` scripts
- Post-process the results using Paraview

# Discretization of the grad operator



$$R_i^v = \frac{\partial(v_j v_i)}{\partial x_j} - \frac{\partial}{\partial x_j} \left[ (\nu + \nu_t) \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

Computing the grad operator in cell O (e.g. grad(p)):

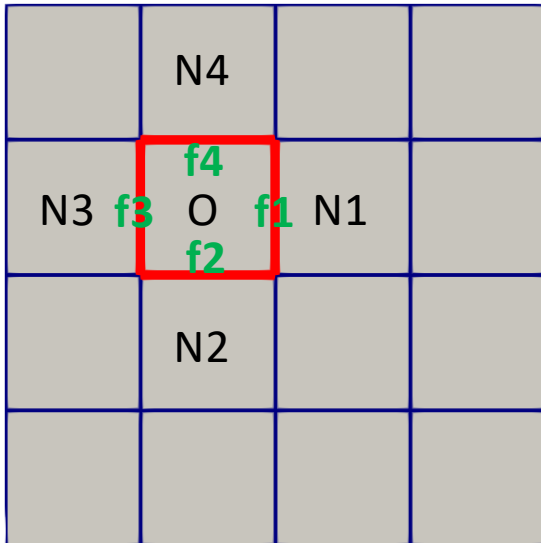
$$\int_{\Omega} \frac{\partial p}{\partial x_i} d\Omega = \int_S p n_i dS = \sum_f p^f n_i^f \Delta S^f$$

**Gauss**                      **Discretization**  
**Divergence**  
**Theorem**

$$p^{fO, N_1} = w^f p^O + (1 - w^f) p^{N_1}$$

**Interpolation**

# Discretization of convection terms: 1st/2<sup>nd</sup> order schemes



$$R_i^v = \frac{\partial(v_j v_i)}{\partial x_j} - \frac{\partial}{\partial x_j} \left[ (\nu + \nu_t) \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

Physical meaning of the convection term?

$$\int_{\Omega} \frac{\partial(v_j v_i)}{\partial x_j} d\Omega = \int_S v_i (v_j n_j) dS = \sum_f v_i^f (v_j n_j)^f \Delta S^f$$

Interpolation??

??
linear

2nd order. In OpenFOAM: linearUpwind

$$\overbrace{v_i^f = v_i^O}$$

1st order. In OpenFOAM: upwind

$$\frac{\partial v_i}{\partial x_j} = \frac{\int_{\Omega} \frac{\partial v_i}{\partial n_j} d\Omega}{\int_{\Omega} d\Omega} = \frac{\int_S v_i n_j dS}{\int_{\Omega} d\Omega} = \frac{\sum_f v_i^f n_j^f \Delta S^f}{\Omega^O}$$

linear

How to compute grad(U)?

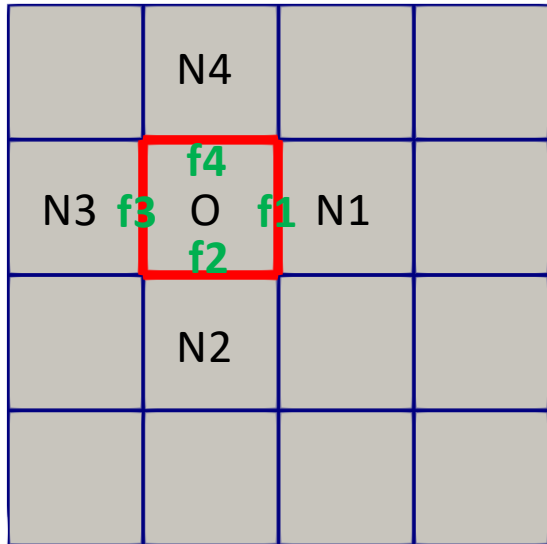


# Discretization of diffusion terms (1)

$$R_i^v = \frac{\partial(v_j v_i)}{\partial x_j} - \frac{\partial}{\partial x_j} \left[ (\nu + \nu_t) \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

$$- \int_{\Omega} \frac{\partial}{\partial x_j} \left[ \nu_{Eff} \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] d\Omega = - \int_S \left[ \nu_{Eff} \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] n_j dS =$$

$$- \underbrace{\int_S \nu_{Eff} \frac{\partial v_i}{\partial x_j} n_j dS}_{D1} - \underbrace{\int_S \nu_{Eff} \frac{\partial v_j}{\partial x_i} n_j dS}_{D2}$$

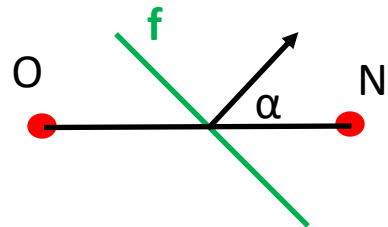


$$D1 = - \int_S \nu_{Eff} \frac{\partial v_i}{\partial x_j} n_j dS = - \sum_f \nu_{Eff}^f \left( \frac{\partial v_i}{\partial x_j} n_j \right)^f \Delta S^f$$

$$\left( \frac{\partial v_i}{\partial x_j} n_j \right)^f = \frac{v_i^N - v_i^O}{\Delta^{NO}}$$

linear

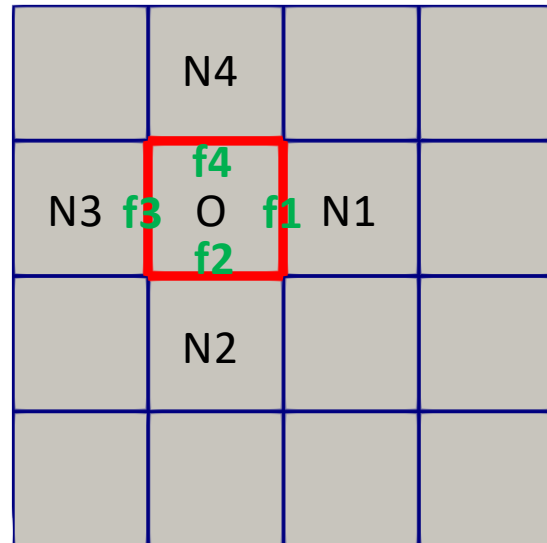
uncorrected. Assumes zero (or non-significant) non-orthogonality



Angle  $\alpha$ : defines the non orthogonality of each faces  $\rightarrow$  checkMesh

- Correction for non-orthogonality: **corrected** surface normal Gradient (snGrad) scheme
- On meshes with very high non-orthogonality: **limited 0.3333** snGrad scheme

# Additional discretization for the momentum diffusion term



$$R_i^v = \frac{\partial(v_j v_i)}{\partial x_j} - \frac{\partial}{\partial x_j} \left[ (\nu + \nu_t) \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] + \frac{\partial p}{\partial x_i} = 0$$

$$- \int_{\Omega} \frac{\partial}{\partial x_j} \left[ \nu_{Eff} \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] d\Omega = - \int_S \left[ \nu_{Eff} \left( \frac{\partial v_i}{\partial x_j} + \frac{\partial v_j}{\partial x_i} \right) \right] n_j dS =$$

$$- \underbrace{\int_S \nu_{Eff} \frac{\partial v_i}{\partial x_j} n_j dS}_{D1} - \underbrace{\int_S \nu_{Eff} \frac{\partial v_j}{\partial x_i} n_j dS}_{D2}$$

$$D2 = - \int_S \nu_{Eff} \frac{\partial v_j}{\partial x_i} n_j dS = - \sum_f \nu_{Eff}^f \left( \frac{\partial v_j}{\partial x_i} \right)^f n_j^f \Delta S^f$$

$$\left( \frac{\partial v_j}{\partial x_i} \right)^f = w^f \frac{\partial v_j}{\partial x_i} \Big|_O + (1 - w^f) \frac{\partial v_j}{\partial x_i} \Big|_N$$

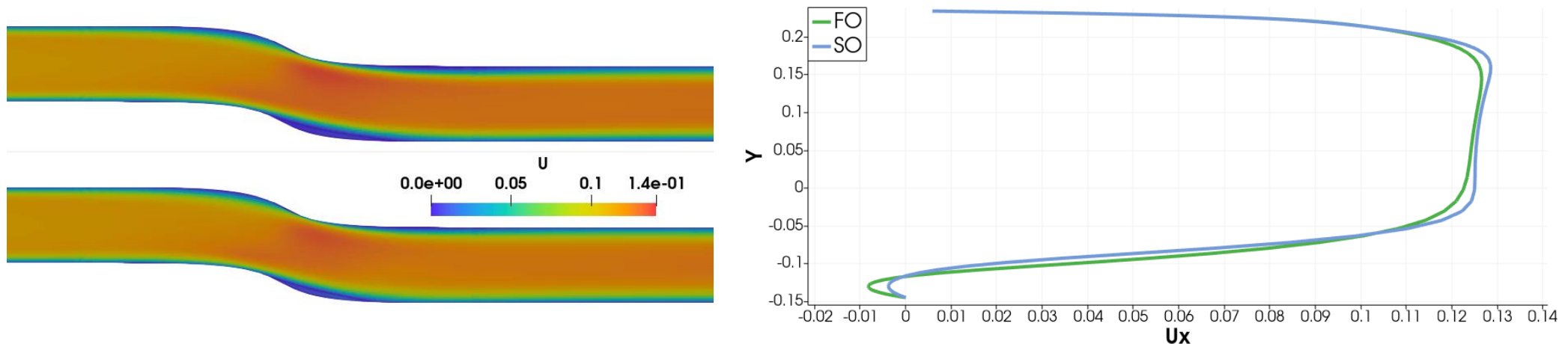
**Linear interpolation**

## Run with a first order scheme and compare

```
>> cd 2024_06_OF_training_EuroCC_Greece/02-firstOrder
```

- Main change in `system/fvSchemes`
- See differences with the second order run by executing  

```
>> diff system/fvSchemes ../01-secondOrder/system/fvSchemes
```
- Run the case and compare with the results from the second order run (total pressure losses, forces, etc)
- Compare the flow fields of the two runs in Paraview



## Using *functions* through controlDict (1)

Entries in `system.functions` can be used:

- To execute auxiliary code at the end of each iteration and/or run
- To compute new fields for post-processing (e.g. total pressure)
- To get useful post-processing content to files (e.g. residual history, flow rate, etc)
- Basic controls/entries for `functions` can be found [here](#)
- Useful examples of already setup functions can be found under `$FOAM_ETC/caseDicts/postProcessing`

## Using *functions* through controlDict (2)

As an exercise, compute the forces on the *lower* and *upper* part of the duct

- Locate **forces** in `$FOAM_ETC/caseDicts/postProcessing`
- Copy a setup for incompressible flows and put it under **system**
- Replace the **patches** list with the one relevant to our case
- Include the additional function to `controlDict.functions`
- Execute only this function without re-running, by using  
`>> mpirun -np 4 simpleFoam -parallel -postProcess -func "forces" -latestTime`



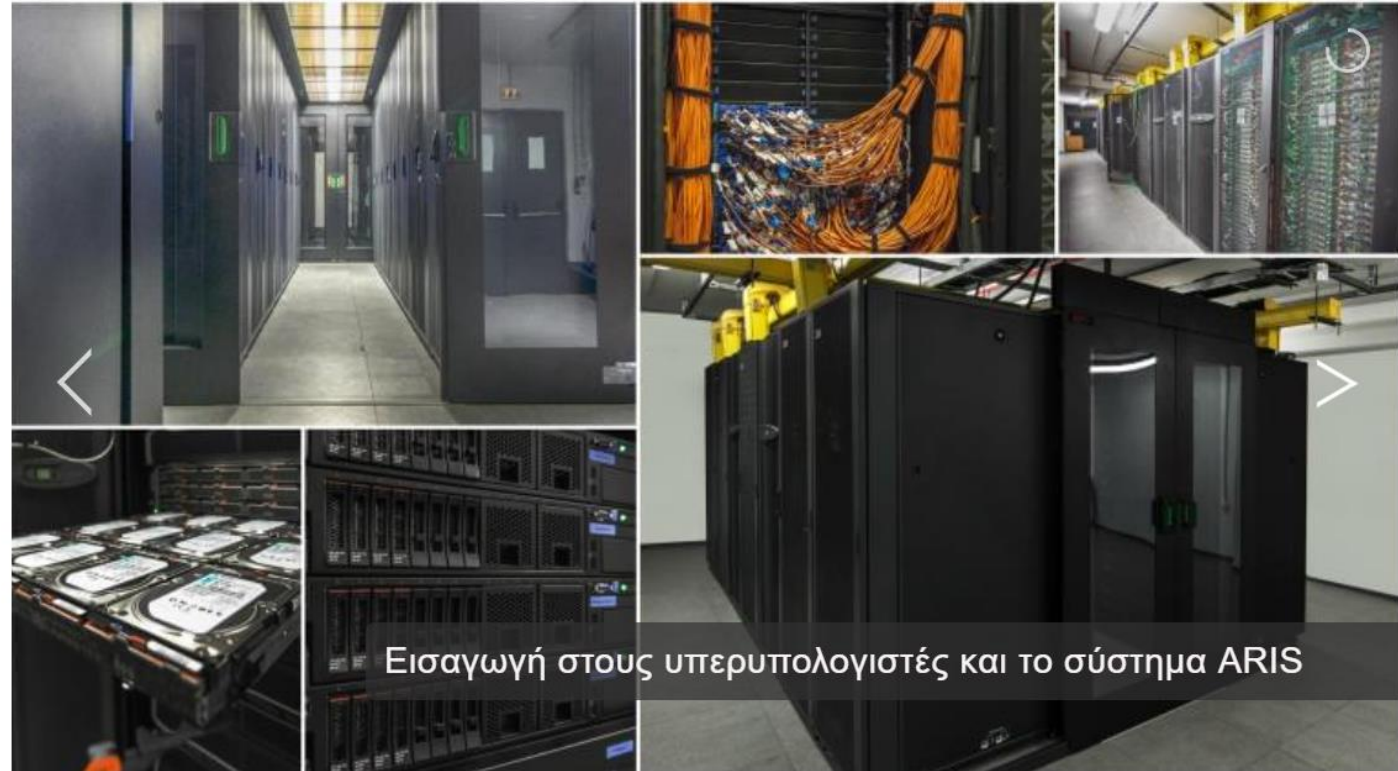
When in doubt about the possible flags available to an executable, use the `-help` option, e.g  
`>> simpleFoam -help`

## Using *functions* through controlDict (3)

- Run the case with first and second order accuracy for the convection terms (see comments for the [upwind](#) and [linearUpwind](#) schemes in slide 13)
- Compute the difference of pt losses between the Inlet and Outlet patches for the two cases

# The ARIS HPC system

- The Greek national HPC system
- Documentation in <https://doc.aris.grnet.gr>
- Multiple partitions
  - Thin: 8520 CPU cores
  - GPU: 88 NVidia K40 GPUs
  - ML: 8 NVidia V100 GPUs
  - FAT: CPU nodes with an emphasis on abundance of memory
- Apply for access: [https://www.hpc.grnet.gr/access\\_policy](https://www.hpc.grnet.gr/access_policy)
- Instructions on how to submit a job: <https://doc.aris.grnet.gr/run>



960 CPUs running for 24h =  
23040 CPUhours

